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弧面凸轮单侧面加工多重包络原理与刀位控制方法研究

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摘要:为提高弧面凸轮廓面加工精度、减小加工过程中刀具误差对凸轮廓面法向误差的影响,分析了弧面凸轮结构特点和现有加工方式存在的问题,提出单侧面加工多重包络原理,并进行了实例仿真计算。利用空间啮合原理和旋转变换矩阵,根据多重包络原理推导出凸轮实际廓面方程,研究了多重包络原理的刀位补偿和刀位控制方法。仿真计算和分析表明,利用单侧面加工多重包络原理可显著减小凸轮廓面法向误差、提高凸轮加工精度。

关键词:弧面分度凸轮; 多重包络; 刀位补偿; 误差计算

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Multi-envelope Principle and Tool Position Control Method of One-side Machining of Globoidal Indexing Cam

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Abstract: In order to improve the machining accuracy of the cam profile surface and reduce the influence of tool errors on the normal error of the cam profile during the machining process, by analyzing the structural characteristics of the cam profile indexing cam and the existing problems in the machining method, and reducing the extreme difference between the actual profile and the theoretical profile obtained by processing, the principle of multiple envelopes for one-side processing was proposed. The effectiveness of the method was demonstrated and a simulation calculation was carried out. The theoretical research was based on the cam profile equation of the arc surface indexing of the set coordinate system, using spatial meshing principle and rotation transformation matrix, according to the principle of one-side machining and multiple envelopes, the actual contour equation after the machining of the globular indexing cam was deduced and calculated, the tool position compensation and tool control method of one-side machining multiple envelope principle was researched. The problem of tool position control based on the principle of multiple envelopes for one-side machining of a globular indexing cam was solved. Matlab simulation results showed that the method can significantly reduce the normal error of the cam profile surface and improve the machining accuracy of the cam, and it had important practical application value.

Key words: globoidal indexing cam; multiple envelope; tool position compensation; error calculation

0 引言

弧面分度凸轮机构因结构紧凑、分度精度高、动力学性能优良等特点,在自动化机械和装备领域应用较为广泛^[1]。弧面凸轮机构由基体为圆弧回转体的主动凸轮和装有滚子的从动盘构成。该机构运转平稳,与其他间歇机构相比,其噪声和振动都较小,故多用于高速机床、包装机和多色印刷机

上^[2-5]。

目前,弧面凸轮加工可分为两大类^[6]:第1类是等径加工法,即采用与滚子尺寸相同的刀具,使刀具与工件再现机构啮合运动,从而进行弧面分度凸轮的加工。加工时除常规的三轴联动外,还需要工件旋转与刀具摆动,一般采用五轴数控机床^[7]。这种方法运算简单、容易实现、没有理论误差^[8-9],但由于需要特质刀具,故适用范围小,而且刀具在加工过

程中产生的磨损会直接影响凸轮的廓面精度。在实际加工过程中,凸轮槽逆铣一侧的表面加工质量差,也降低了机构精度^[10-12]。第2类是非等径加工方法。非等径加工方法一般分为3种:①使用端面铣刀或者球铣刀进行点位加工的仿自由曲面法,这种方法将凸轮廓面视为空间自由曲面,加工后机构失去包络性质,而且效率低、精度低。②两重包络法,其优点是刀具半径可选范围较大,理论误差为零,但需要处理的数据量大,加工效率不高^[13-15]。③刀位补偿法,使刀具通过一定的补偿方法与凸轮再现机构啮合运动,从而进行工件加工,该方法加工速度快,刀径可选择范围大^[16-17]。

基于上述加工方法,研究既满足相应工艺精度要求同时又具有良好经济性的加工方法具有重要意义和应用价值。本文提出单侧面加工多重原理,并进行实例仿真计算和分析,以验证方法的有效性。

1 弧面凸轮理论廓面方程

建立图1所示的弧面分度凸轮坐标系,其中 $oxyz$ 为固定坐标系,与机架固结; $o_1x_1y_1z_1$ 和 $o_2x_2y_2z_2$ 为动坐标系,分别与凸轮和从动件固结。

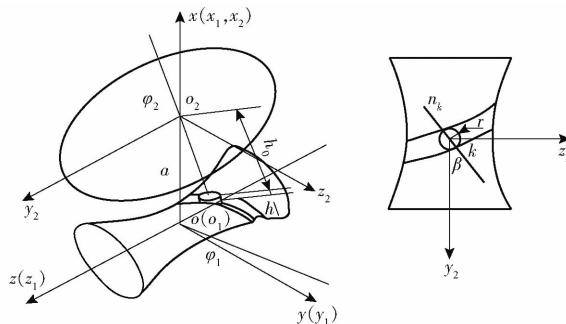


图1 弧面分度凸轮坐标系

Fig. 1 Goboidal indexing cam coordinate system

利用空间啮合原理和旋转变换矩阵^[18]推导出弧面凸轮的理论工作廓面为

$$\left\{ \begin{array}{l} x_0 = [a - \cos\varphi_2(h_0 + h) - r\sin\varphi_2\cos\beta]\cos\varphi_1 - \\ \quad r\sin\varphi_1\sin\beta \\ y_0 = [a - \cos\varphi_2(h_0 + h) - r\sin\varphi_2\cos\beta]\sin\varphi_1 - \\ \quad r\cos\varphi_1\sin\beta \\ z_0 = 0 - \sin\varphi_2(h_0 + h) + r\cos\varphi_2\cos\beta \end{array} \right. \quad (1)$$

凸轮机构啮合方程为

$$\beta = \arctan \frac{\frac{d\varphi_2}{d\varphi_1}(h + h_0)}{a - \cos\varphi_2(h + h_0)} \quad (2)$$

式中 a —中心距 φ_1 —凸轮角位移

φ_2 —从动件角位移

h —任意截面到滚子内侧端面的距离
 h_0 —滚子内侧端面到从动盘回转中心的距离
 r —滚子半径
 β —凸轮上廓面理论接触角

2 单侧面加工多重包络原理

由于缺乏整体性导致单侧面加工凸轮廓面误差较大^[19],影响廓面的整体加工精度;本文提出的单侧面加工多重包络原理通过多次分析实际接触线和理论接触线的位置关系,保证加工后的廓面误差最小,使得加工精度进一步提高。

如图2所示,一重包络加工后通过分析凸轮廓面法向最大误差以确定二重包络加工时非等径刀具位置,二重包络后再次分析凸轮廓面法向最大误差以确定三重包络加工时非等径刀具位置;通过多次改变非等径刀具位置来实现弧面分度凸轮的多重包络加工。

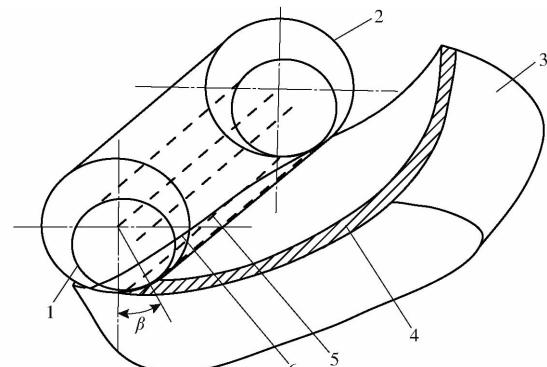


图2 多重包络原理加工过程

Fig. 2 Multi-envelope principle processing process

1. 非等径刀具
2. 等径刀具
3. 零件本体
4. 加工余量
5. 理论接触线
6. 实际接触线

2.1 多重包络刀位补偿

弧面凸轮廓面和滚子的理论接触线在停歇段时为直线,其他位置时均为图3所示的空间曲线 MN ^[20-21],且随着机构的运动,啮合点的变化, MN 也随之改变;同理将刀具与凸轮加工廓面相互接触的空间曲线 PQ 称为实际接触线。

由图3可看出,无论怎样调整刀具位置都无法使得两曲线完全重合,肯定会产生理论误差^[22],若使刀具的中心始终位于滚子宽度一半($H/2$)的接触角 β_0 上,且刀具中心与滚子理论中心相距 ΔR ($\Delta R = r - r_t$),这样虽然两条空间曲线 MN 和 PQ 仍在不同柱面上,但在滚子接触长度一半处两曲线相切于点A,此时在切点A处凸轮的廓面法向误差为 $\Delta n_a = 0$,两接触线彼此之间的距离最近,误差最小^[23]。

通过这种刀位补偿法即可实现廓面误差最小的目的,同时这种方法使得误差补偿方向固定,并且有

有效地避免了廓面误差分布的不规律性^[24],图4为刀具与滚子相互位置关系, o_t 为刀具中心。

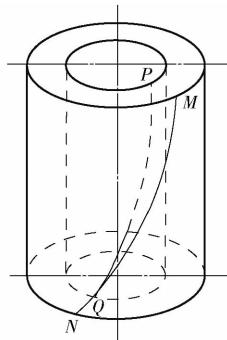


图3 接触线位置关系

Fig. 3 Contact line position relationship

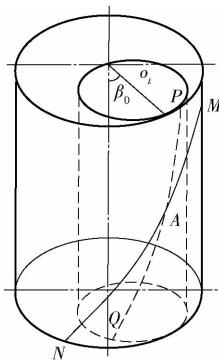


图4 一重包络刀具与滚子位置关系

Fig. 4 Position relationship between single envelope cutter and roller

由于一重包络加工使得滚子接触长度一半处凸轮廓面法向误差为零,但是在滚子两端处误差较大。因此二重包络时使刀具中心始终位于滚子宽度1/4($H/4$)的接触角 β_1 上,且刀具中心与滚子理论中心相距 ΔR ,此时在滚子接触长度四分之一处两接触线相切于点B,且在切点B处凸轮的廓面法向误差 $\Delta n_b = 0$,如图5所示。

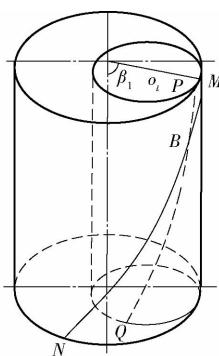


图5 二重包络刀具与滚子位置关系

Fig. 5 Position relationship between double envelope cutter and roller

三重包络时将刀具中心置于滚子宽度四分之三($3H/4$)的接触角 β_2 上,且刀具中心与滚子理论中心相距 ΔR ,此时在滚子接触长度四分之三处两接触

线相切于点C,且在切点C处凸轮廓面法向误差 $\Delta n_c = 0$,如图6所示;通过多重包络加工使得滚子两端处误差得到有效减小,提高了弧面分度凸轮的整体精度。

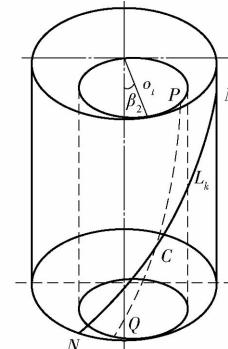


图6 三重包络刀具与滚子位置关系

Fig. 6 Position relationship between triple envelope cutter and roller

2.2 多重包络刀位控制

一重包络加工时在中心距、从动件运动规律等参数不变的情况下,将刀具中心 o_t 置于滚子接触长度一半($H/2$)的接触角 β_0 上,且与滚子中心 o_t 相距 ΔR ,如图7所示。

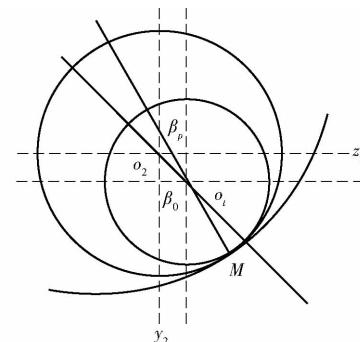


图7 一重包络刀位控制方法

Fig. 7 Single envelope tool position control method

此时在 $o_2x_2y_2z_2$ 坐标系中,刀具任意截面($h + h_0$)内,刀具与凸轮啮合点M的矢径为

$$\mathbf{r}_{M1} = \begin{bmatrix} -(h + h_0) \\ \Delta R \cos \beta_0 + r \cos \beta_p \\ \Delta R \sin \beta_0 + r \sin \beta_p \end{bmatrix} \quad (3)$$

式中 β_p ——凸轮与滚子实际接触角

此时弧面凸轮实际工作廓面为

$$\left\{ \begin{array}{l} x_1 = [a - \cos \varphi_2 (h_0 + h) - (r \cos \beta_p + \Delta R \cos \beta_0) \cdot \sin \varphi_2] \cos \varphi_1 - (r \sin \beta_p + \Delta R \sin \beta_0) \sin \varphi_1 \\ y_1 = [a - \cos \varphi_2 (h_0 + h) - (r \cos \beta_p + \Delta R \cos \beta_0) \cdot \sin \varphi_2] \sin \varphi_1 + (r \sin \beta_p + \Delta R \sin \beta_0) \cos \varphi_1 \\ z_1 = 0 - \sin \varphi_2 (h_0 + h) + (r \cos \beta_p + \Delta R \cos \beta_0) \cdot \cos \varphi_2 \end{array} \right. \quad (4)$$

其中

$$\beta_p = \arctan \frac{\frac{d\varphi_2}{d\varphi_1}(h + h_0) - \Delta R \sin \varphi_2 \sin \beta_0}{a - \cos \varphi_2(h + h_0) - \Delta R \cos \beta_0 \sin \varphi_2} \quad (5)$$

二重包络时各参数不变,将刀具中心 o_t 置于滚子接触长度 $1/4(H/4)$ 的接触角 β_1 上,且与滚子中心 o_2 相距 ΔR ,如图 8 所示。

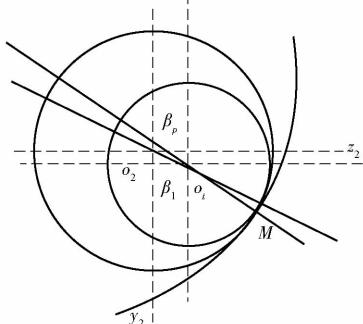


图 8 二重包络刀位控制方法

Fig. 8 Double envelope tool position control method

此时在 $o_2x_2y_2z_2$ 坐标系中,刀具任意截面($h + h_0$)内,刀具与凸轮啮合点 M 的矢径为

$$\mathbf{r}_{M2} = \begin{bmatrix} -(h + h_0) \\ \Delta R \cos \beta_1 + r \cos \beta_p \\ \Delta R \sin \beta_1 + r \sin \beta_p \end{bmatrix} \quad (6)$$

弧面凸轮的实际工作廓面为

$$\left\{ \begin{array}{l} x_2 = [a - \cos \varphi_2(h_0 + h) - (r \cos \beta_p + \Delta R \cos \beta_1) \cdot \sin \varphi_2] \cos \varphi_1 - (r \sin \beta_p + \Delta R \sin \beta_1) \sin \varphi_1 \\ y_2 = [a - \cos \varphi_2(h_0 + h) - (r \cos \beta_p + \Delta R \cos \beta_1) \cdot \sin \varphi_2] \sin \varphi_1 + (r \sin \beta_p + \Delta R \sin \beta_1) \cos \varphi_1 \\ z_2 = 0 - \sin \varphi_2(h_0 + h) + (r \cos \beta_p + \Delta R \cos \beta_1) \cdot \cos \varphi_2 \end{array} \right. \quad (7)$$

其中

$$\beta_p = \arctan \frac{\frac{d\varphi_2}{d\varphi_1}(h + h_0) - \Delta R \sin \varphi_2 \sin \beta_1}{a - \cos \varphi_2(h + h_0) - \Delta R \cos \beta_1 \sin \varphi_2} \quad (8)$$

三重包络时各参数保持不变,将刀具中心 o_t 置于滚子宽度 $3/4(3H/4)$ 的接触角 β_2 上,且与滚子中心 o_2 相距 ΔR ,如图 9 所示。

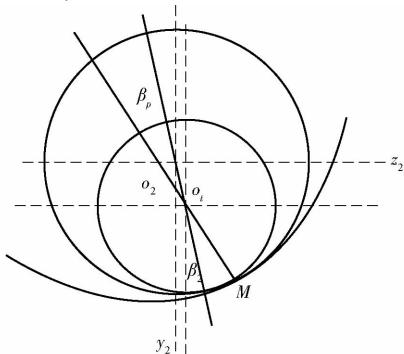


图 9 三重包络刀位控制方法

Fig. 9 Triple envelope tool position control method

此时在 $o_2x_2y_2z_2$ 坐标系中,刀具任意截面($h + h_0$)内,刀具与凸轮啮合点 M 的矢径为

$$\mathbf{r}_{M3} = \begin{bmatrix} -(h + h_0) \\ \Delta R \cos \beta_2 + r \cos \beta_p \\ \Delta R \sin \beta_2 + r \sin \beta_p \end{bmatrix} \quad (9)$$

弧面凸轮的实际工作廓面为

$$\left\{ \begin{array}{l} x_3 = [a - \cos \varphi_2(h_0 + h) - (r \cos \beta_p + \Delta R \cos \beta_2) \cdot \sin \varphi_2] \cos \varphi_1 - (r \sin \beta_p + \Delta R \sin \beta_2) \sin \varphi_1 \\ y_3 = [a - \cos \varphi_2(h_0 + h) - (r \cos \beta_p + \Delta R \cos \beta_2) \cdot \sin \varphi_2] \sin \varphi_1 + (r \sin \beta_p + \Delta R \sin \beta_2) \cos \varphi_1 \\ z_3 = 0 - \sin \varphi_2(h_0 + h) + (r \cos \beta_p + \Delta R \cos \beta_2) \cdot \cos \varphi_2 \end{array} \right. \quad (10)$$

其中

$$\beta_p = \arctan \frac{\frac{d\varphi_2}{d\varphi_1}(h + h_0) - \Delta R \sin \varphi_2 \sin \beta_2}{a - \cos \varphi_2(h + h_0) - \Delta R \cos \beta_2 \sin \varphi_2} \quad (11)$$

3 弧面凸轮廓面法向误差模型

理想情况下凸轮廓面与加工后凸轮的实际廓面之间的对比即法向误差模型,对比每一刻的刀具切削线与理想接触线,即实际廓面与理论廓面的法向误差^[25-26]。

如图 10 所示,过实际工作廓面上的一点 m 作法线,与理论工作廓面交于点 n , Δn 为凸轮廓面的法向误差。

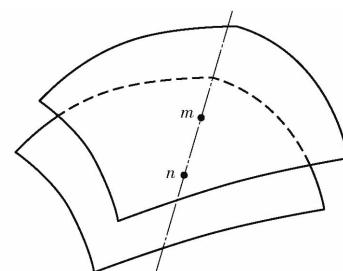


图 10 法向误差计算

Fig. 10 Normal error calculation

在定坐标 oxy 中,过点 m 的单位法向矢量为

$$\mathbf{n}_{m0} = \begin{bmatrix} x_{nm1} \\ y_{nm1} \\ z_{nm1} \end{bmatrix} = \begin{bmatrix} -\sin \varphi_2 \cos \beta_p \\ \sin \beta_p \\ -\cos \varphi_2 \cos \beta_p \end{bmatrix} \quad (12)$$

在 $o_1x_1y_1z_1$ 中该法线矢量为

$$\mathbf{n}_{m1} = \mathbf{M}_{10} + \mathbf{n}_{m0} \quad (13)$$

式中 \mathbf{M}_{10} ——定坐标系 oxy 到动坐标系 $o_1x_1y_1z_1$ 的交换矩阵

过点 m 的法线方程为

$$\frac{x - x_m}{x_{nm1}} = \frac{y - y_m}{y_{nm1}} = \frac{z - z_m}{z_{nm1}} \quad (14)$$

由凸轮廓面形成原理可知,实际工作廓面中点 m 处于啮合状态时,理论工作轮廓上的点 n 尚未处于或已经越过啮合状态,未知量为参数 φ_1 及 h 。

联立式(1)、(14)可得一组非线性方程,使用牛顿迭代法求解^[27]。

将求解的法线交点 n 所对应的 φ_n 及 h_n 代入式(1)可得交点 n 的坐标值,其法向误差为

$$\Delta n = \sqrt{(x_m - x_n)^2 + (y_m - y_n)^2 + (z_m - z_n)^2} \quad (15)$$

4 计算实例

现加工一右旋蜗形凸轮,其滚子半径 $r = 20$ mm,中心距 $a = 180$ mm,滚子内侧端面到从动盘回转中心的距离 $h_0 = 20$ mm,滚子宽度 $H = 20$ mm,加工刀具半径 $r_t = 17$ mm,运动规律为修正正弦加速运动规律。

刀具半径补偿 $\Delta R = 3$ mm 时,一重包络加工后凸轮廓面法向误差为 Δn_1 ,分布见图 11。若使用等径法加工,令其加工后凸轮廓面法向误差为 Δn_0 ,由于刀具的误差对凸轮廓面法向误差的影响是 1:1,则凸轮廓面任意位置的法向误差 $\Delta n_0 = \Delta R = 3$ mm,而一重包络加工后,凸轮廓面法向误差在滚子接触长度一半处($H/2$)为零,其两端误差较大;且法向误差在滚子中点两侧规律分布。

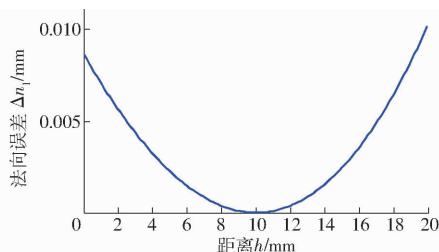


图 11 一重包络法向误差沿滚子轴截面分布

Fig. 11 Single envelope normal error distributed along roller shaft section

二重包络加工后的凸轮廓面法向误差为 Δn_2 ,如图 12 所示,相较于一重包络而言二重包络大大降低了凸轮廓面一端的误差,且在滚子接触 1/4

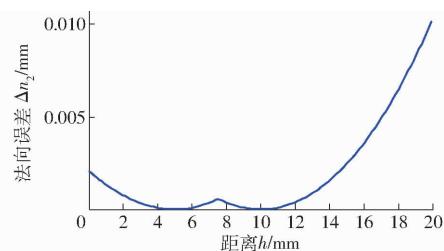


图 12 二重包络法向误差沿滚子轴截面分布

Fig. 12 Double envelope normal error distributed along section of roller shaft

($H/4$) 处误差为零。

三重包络加工后凸轮廓面法向误差为 Δn_3 ,如图 13 所示,可以看出经过三重包络加工后凸轮廓面最大法向误差 $\Delta n_{3\max} = 0.002712$ mm,且在滚子接触长度 $3/4(3H/4)$ 处法向误差为 0。

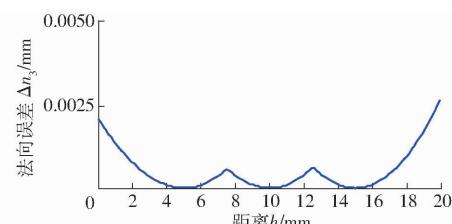


图 13 三重包络法向误差沿滚子轴截面分布

Fig. 13 Triple envelope normal error distributed along section of roller shaft

综合分析可得,利用本文所提出的多重包络原理可显著减小凸轮廓面法向误差,减小刀具对凸轮廓面的影响,提高弧面凸轮的加工精度。

5 结束语

通过对刀位补偿法存在问题进行分析,根据接触线关系提出了单侧面加工多重包络原理,利用该原理可以显著减小凸轮廓面法向误差,降低刀具误差对凸轮廓面法向误差的影响,使加工误差更小,加工效果更为理想。由于弧面凸轮定位段处滚子与凸轮廓面的接触线为直线,因此定位段处凸轮廓面没有法向误差,机构分度精度和定位精度得到有效保障。

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